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OPTIMAL DESIGN OF AN OPEN-ENDED BRAIDED STENT USING MOPSO

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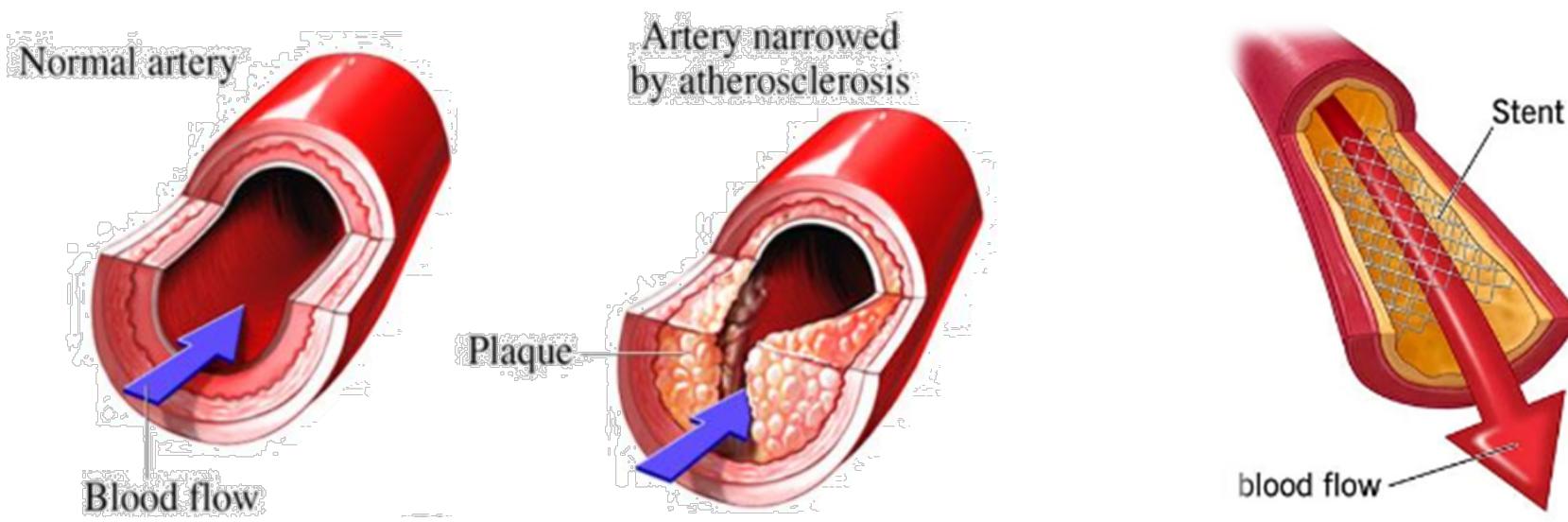
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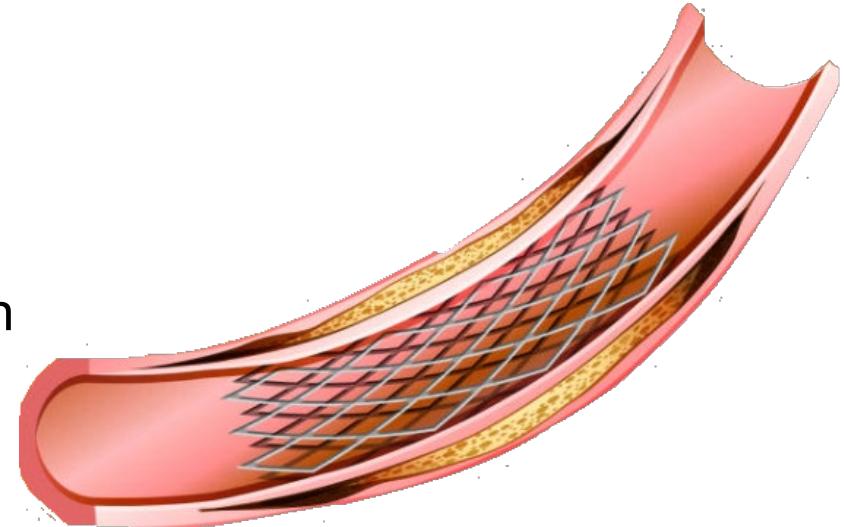
CORONARY ARTERY DISEASE

- Atherosclerosis : fat or lipid substances are deposited on the arterial wall
- Severe atherosclerosis cause coronary artery disease
- effective treatment method is Percutaneous Coronary Intervention (PCI)
- PCI is to unblock and restore blood by placing ~~vascular~~ stent on the stenosis



WHAT IS THE STENT?

- A small mesh-like device made of metal
- Acts as a support or scaffold in keeping the vessel open
- helps to improve blood flow to the heart muscle



Ideal Functions And Mechanical Properties of Stents

High Radial Stiffness **Maintain the vessel open and withstand the periodic load of arteries**

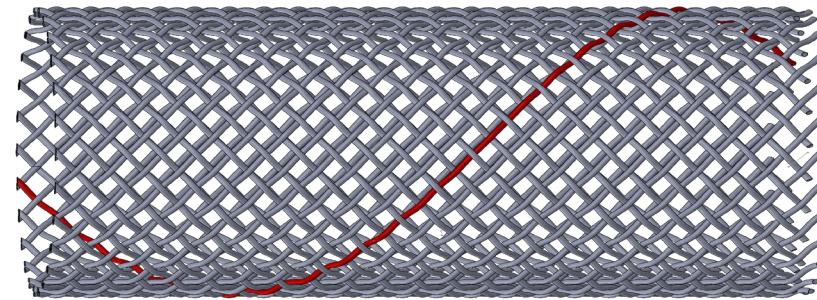
Minimum Foreshortening **Accurately placed in the position**

Maximum Radial Strength **Pushed away the blockage and better anchorage**



PROBLEM STATEMENT AND ASSUMPTIONS

1. Investigate stent based on Odedwab and Clerc study from technical literature
2. Using the MOPSO algorithm to enhance mechanical properties
3. Using AM to reduce mass/cost while maintaining mechanical performance

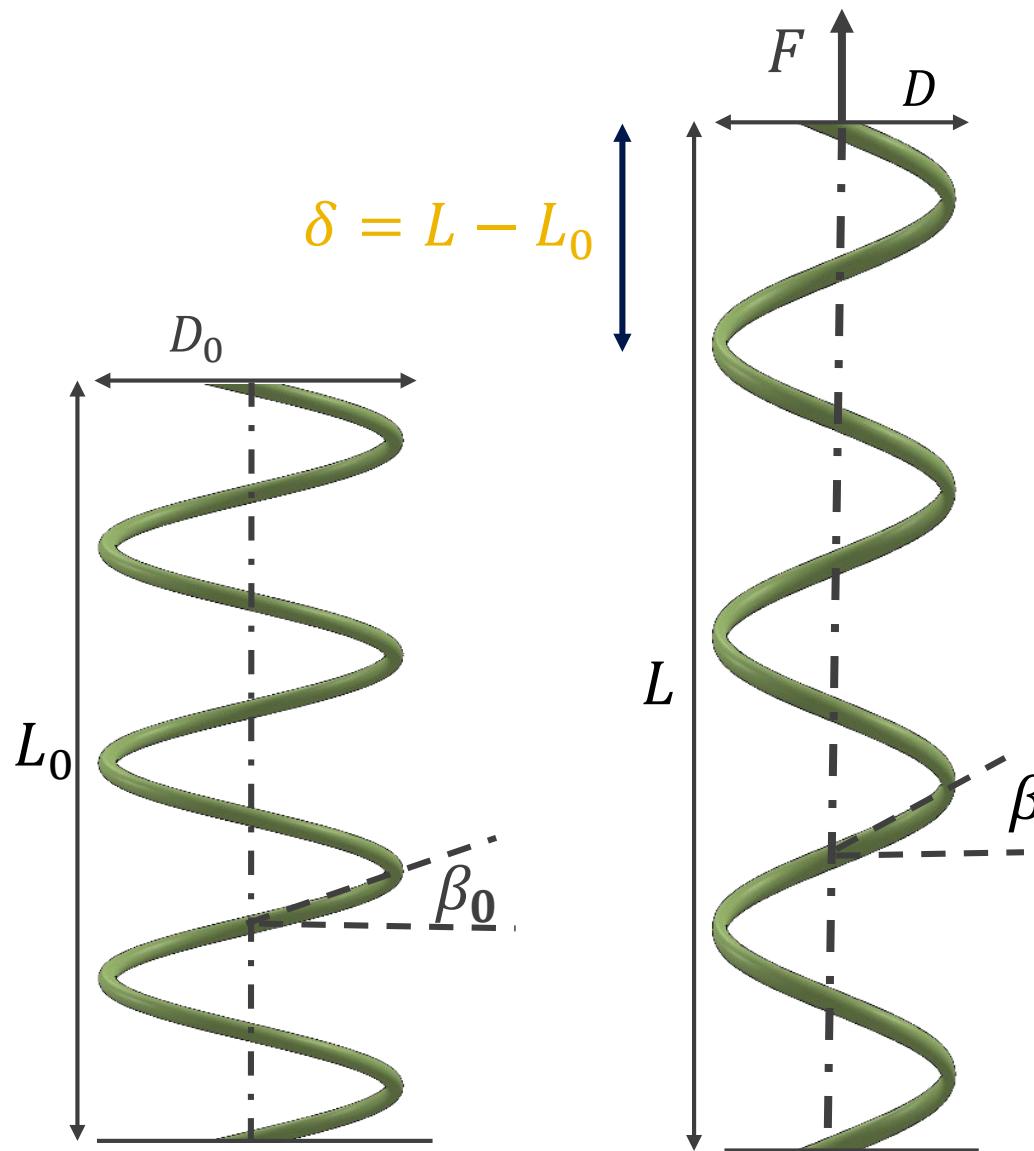


ASSUMPTION ON THE STENT BEHAVIOUR

- a) It acts as a collection of independent open -coiled helical springs .
- b) The ends are fixed against rotating around the longitudinal axis.
- c) It undergoes only elastic deformation



STENT GEOMETRY



INITIAL PITCH

$$p_0 = \pi D_0 \tan \beta_0$$

NUMBER OF COILS

$$c = L_0 / p_0$$

STENT DIAMETER

$$D = \frac{D_0 \cos(\beta)}{\cos(\beta_0)}$$

STENT LENGTH

$$L = \frac{L_0 \sin(\beta)}{\sin(\beta_0)}$$

MATERIAL

Linear elastic steel



MECHANICAL PROPERTIES AND ANALYTICAL MODEL

The axial force F exerted on the stent

$$F = 2N \left[\frac{GI_P}{K_3} \left(\frac{2\sin(\beta)}{K_3} - K_1 \right) - \frac{EI \cdot \tan(\beta)}{K_3} \left(\frac{2\cos(\beta)}{K_3} - K_2 \right) \right]$$

$$K_1 = \frac{\sin(2\beta_0)}{D_0}$$

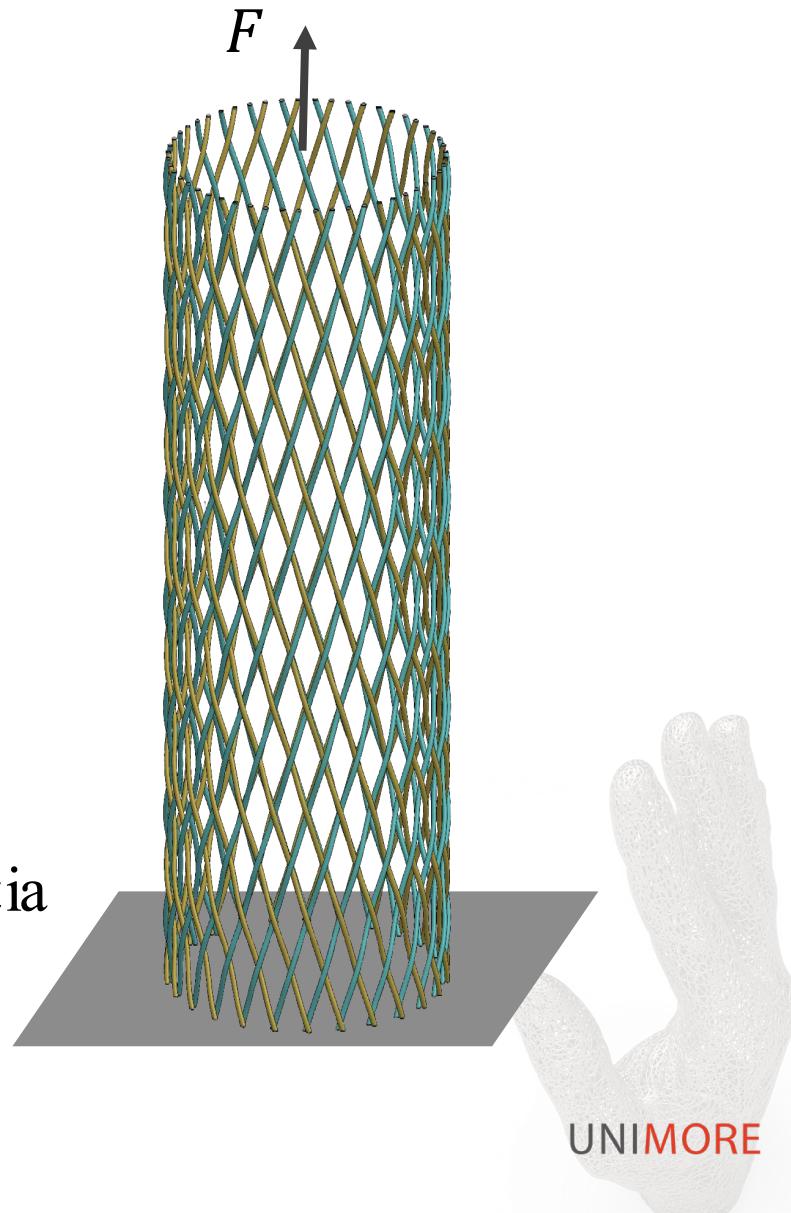
$$K_2 = \frac{2\cos^2(\beta_0)}{D_0}$$

$$K_3 = \frac{D_0}{\cos(\beta_0)}$$

N : the number of wires

I and **I_P** : the moment of inertia and polar moment of inertia

G and **E**: the shear modulus and Young's modulus

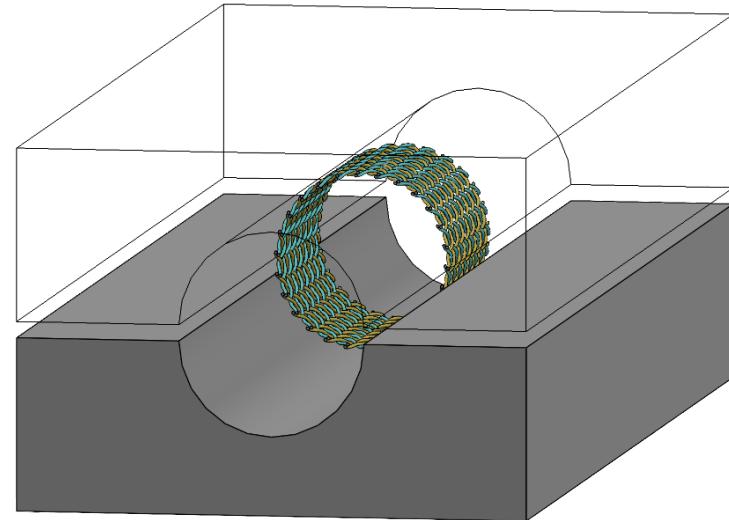


MECHANICAL PROPERTIES

$$dW = F \cdot d\delta \quad \longrightarrow \quad F \cdot d\delta = P\pi DL \frac{dD}{2}$$

$$dW = P\pi DL \frac{dD}{2}$$

P and F perform the equal amount of work



RADIAL PRESSURE

$$P = \frac{2cF}{DL\tan(\beta)}$$

RADIAL FORCE

$$F_R = \frac{2\pi cF}{\tan(\beta)}$$

RADIAL PRESSURE STIFFNESS

$$K_P = \frac{dP}{dD} = \frac{dP}{d\beta} \cdot \frac{d\beta}{dD}$$

$$K_P = \frac{2c}{K_3 \sin(\beta) (DL\tan(\beta))^2} \times \left[2DLN \cdot \tan(\beta) \left(\frac{GI_p}{K_3} \left(\frac{2 \cos(\beta)}{K_3} \right) - \frac{EI}{K_3} \left(\frac{2 \cos(\beta)}{K_3} - \frac{K_2}{\cos^2(\beta)} \right) \right) - F \left(\frac{DL}{\cos^2(\beta)} + K_3 \sin(\beta) \times (\pi c D - L \cdot \tan(\beta)) \right) \right]$$



MECHANICAL PROPERTIES

Shear and bending stress (σ , τ)

$$\tau = \frac{F \cdot \cos(\beta) \cdot D/2 \cdot d/2}{I_p}$$

Twisting moment

$$\sigma = \frac{F \sin(\beta) \cdot D/2 \cdot d/2}{I}$$

Bending moment

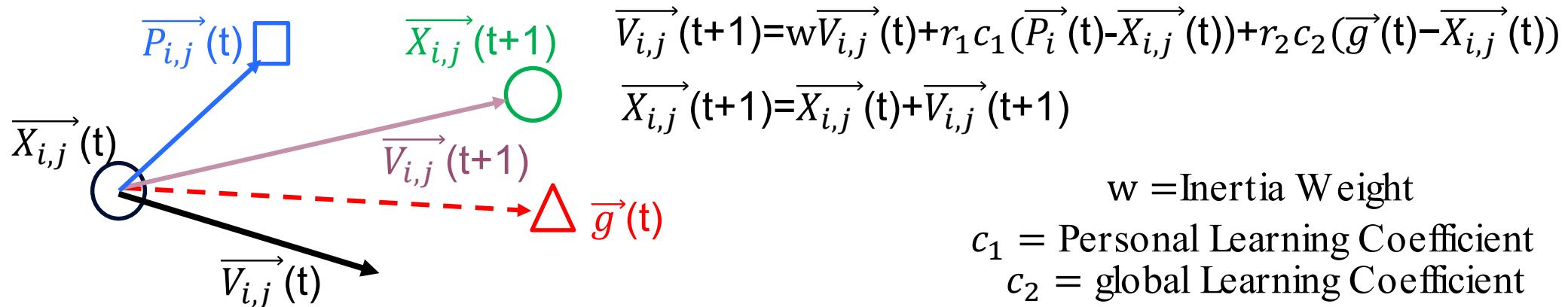


$$\sigma_{eq} = \sqrt{\frac{2\sigma^2 + 6\tau^2}{2}}$$



MULTI OBJECTIVE PARTICLE SWARM OPTIMIZATION

PSO Based on the paradigm of swarm intelligence, introduced by James Kennedy and Russell Eberhart 1995



MOPSO Developed by Carlos et.al 2004

- Like PSO particles are sharing info. & moving towards global best particles and their own personal best memories.
- Unlike PSO there is more than one criterion to determine and define the best global.

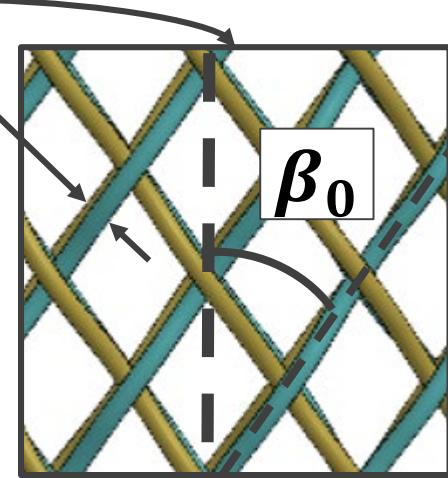
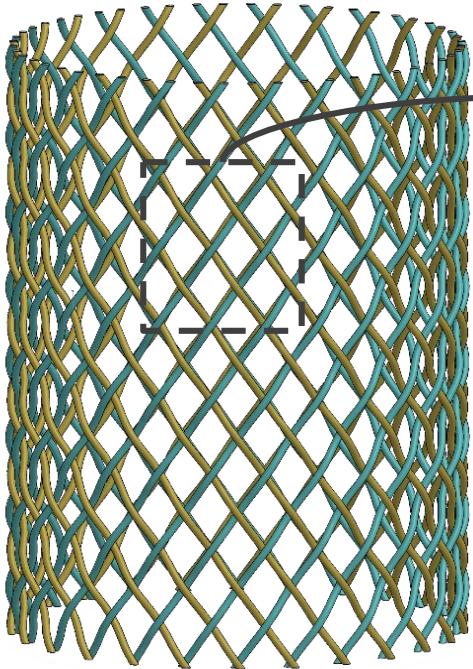
Non-dominated particles



Repository



MULTI OBJECTIVE PARTICLE SWARM OPTIMIZATION



COST FUNCTIONS

- { **Minimize the maximum stent length**
- Maximize the radial Force exerted on the veins**
- Maximize the radial stiffness**

$$V = N \cdot \frac{L_0}{\sin \beta_0} \cdot \pi d^2$$

DESIGN VARIABLES

- N** = Number of wires
 β_0 = Initial pitch angle
 d = Diameter of the wire

CONSTRAINTS

$$V_{Optimized} \leq V_{Jedwab}$$

$$\sigma_{eq} < \text{yield strength}$$

$$\beta_{min} > \arctan \left(\frac{dN}{\pi D} \right)$$



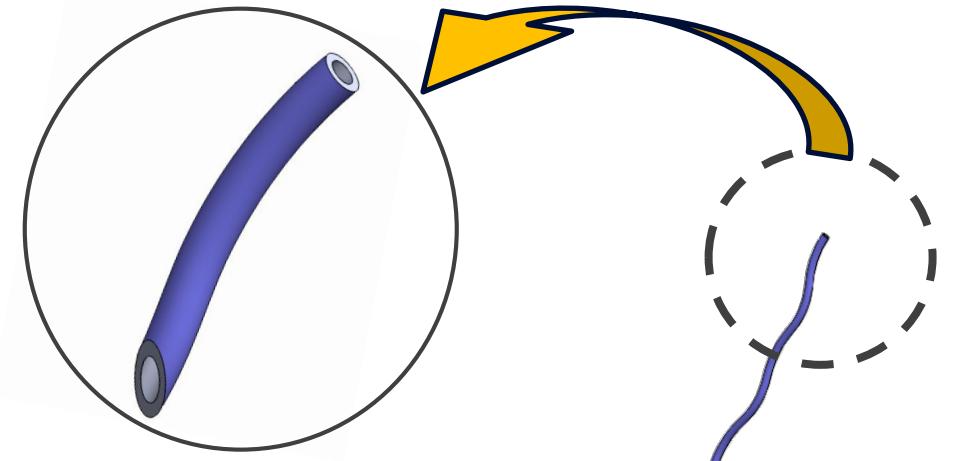
USE OF AM TO GET HOLLOW / LIGHTER STRUCTURE

t = Thickness

$$d_{in} = 2\left(\frac{d}{2} - t\right)$$

$$I_P = \frac{\pi}{32} (d^4 - d_{in}^4)$$

$$I = \frac{\pi}{32} (d^4 - d_{in}^4)$$



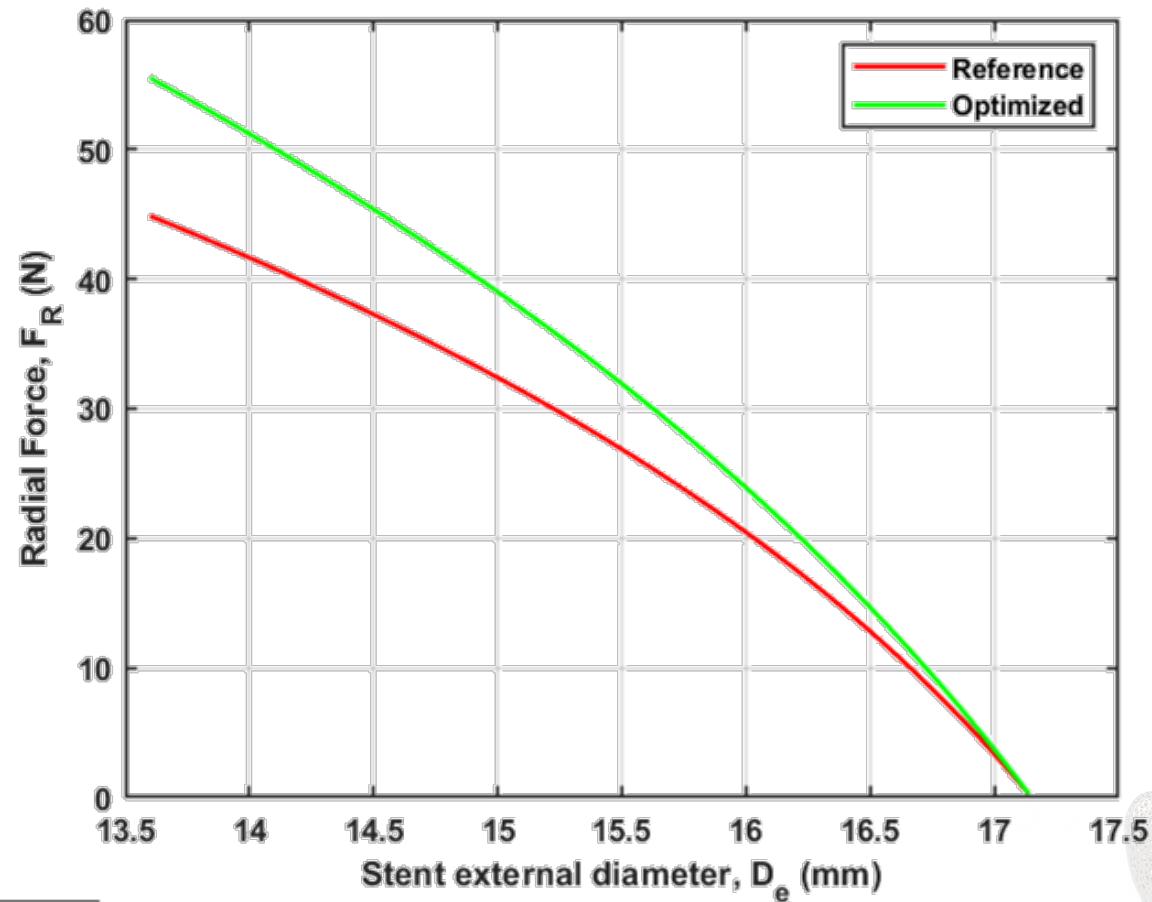
Method	Common material	Resolution
SLM	Stainless steels, aluminium (Al), copper (Cu), iron (Fe), cobalt (Co)-chrome (CR), Ti, Ni-based alloy, and a mixture of different types of particles (Fe, Ni, Cu, and Fe3P)	20 μm
DED	Steel alloys, stainless steel, tool steel, Al, bronze, Co-Cr, (Fe, Ni)-TiC composites and Ti ,	20 μm



RESULTS: RADIAL FORCE

Parameter	value
Initial average stent diameter D_0 (mm)	16.71
Wire diameter d (mm)	0.22
Number of wires N	24
Volume of the stent (mm^3)	673
Initial pitch angle β_0 ($^\circ$)	28.3
Initial stent length L_0 (mm)	87.5
Young's modulus E (MNm^{-2})	206000
Yield stress S_y (MNm^{-2})	2500

Design	N	d	β_0	Volume
Opt.1	18	0.266	31.54	670



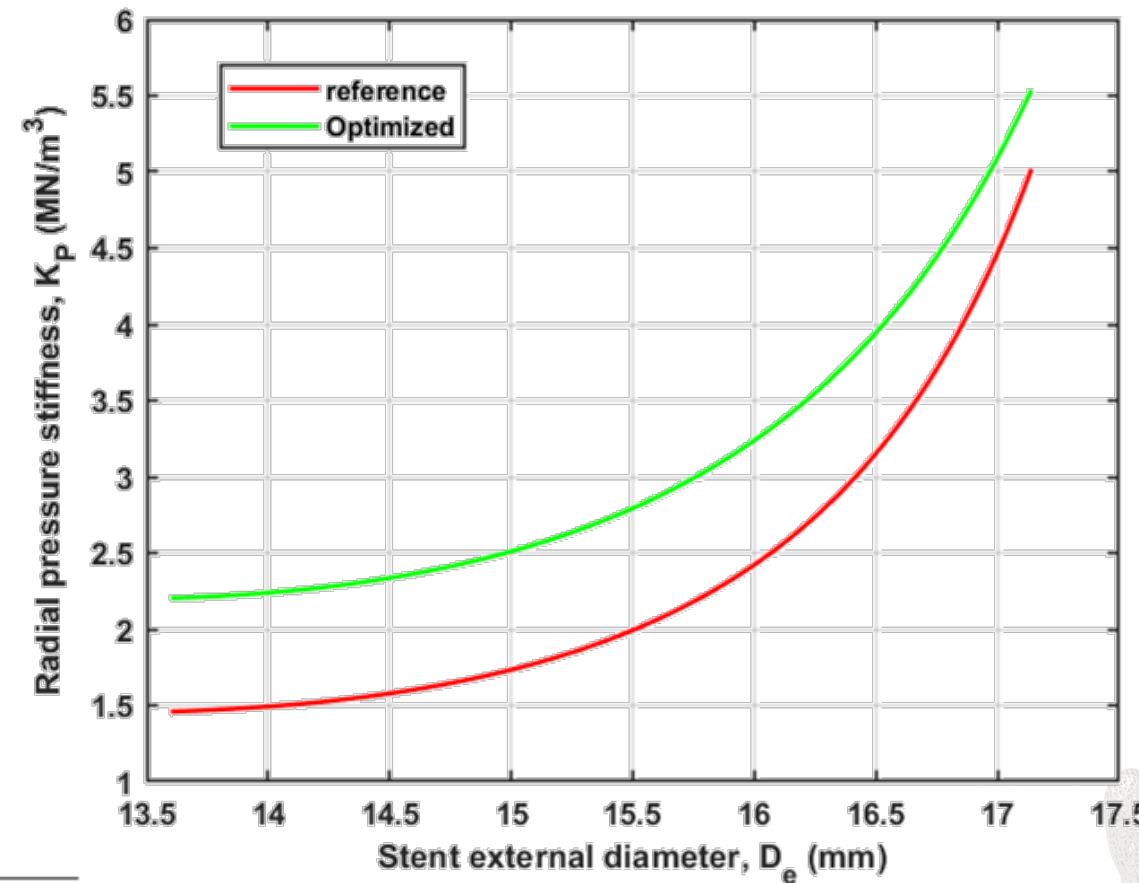
D_0 (fully deployed) $\rightarrow 0.8D_0$



RESULTS: RADIAL PRESSURE STIFFNESS

Parameter	value
Initial average stent diameter D_0 (mm)	16.71
Wire diameter d (mm)	0.22
Number of wires N	24
Volume of the stent (mm^3)	673
Initial pitch angle β_0 ($^\circ$)	28.3
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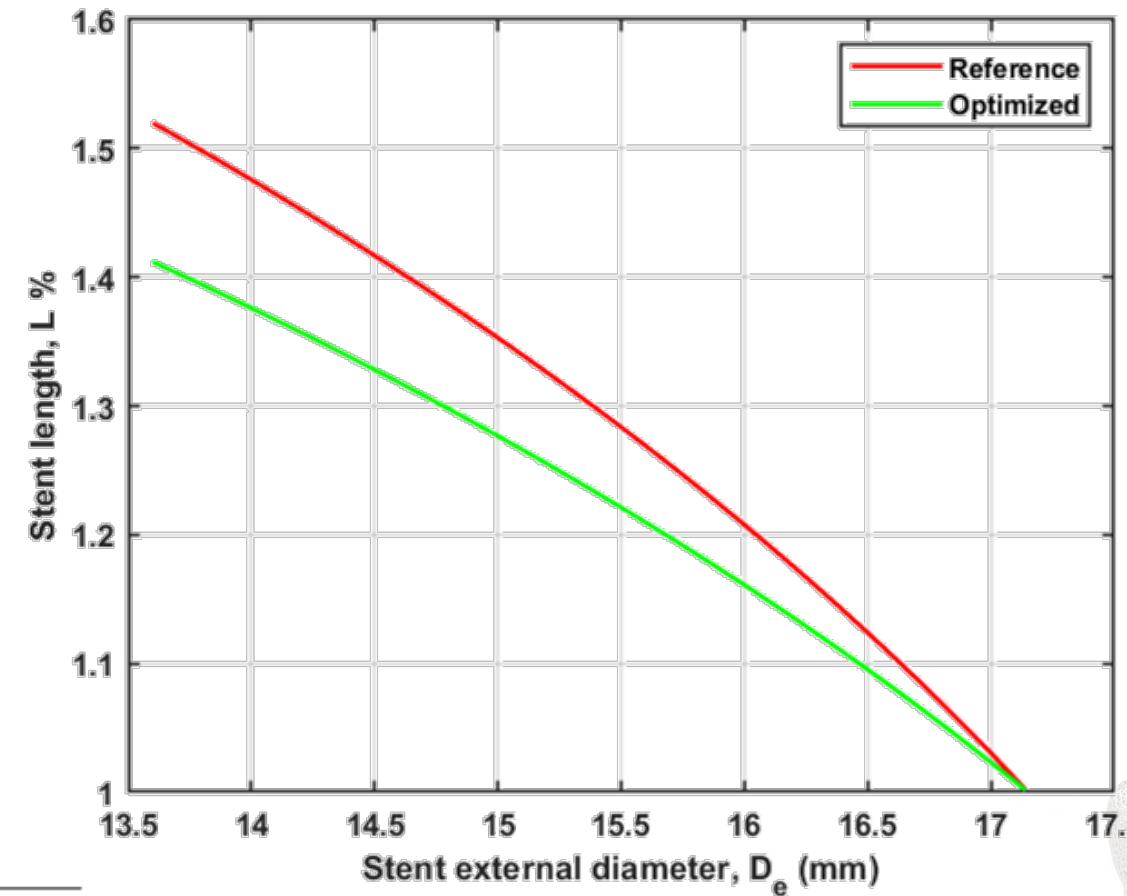
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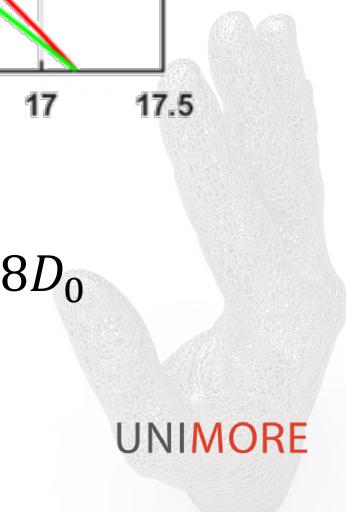
RESULTS: STENT LENGTH

Parameter	value
Initial average stent diameter D_0 (mm)	16.71
Wire diameter d (mm)	0.22
Number of wires N	24
Volume of the stent (mm^3)	673
Initial pitch angle β_0 ($^\circ$)	28.3
Initial stent length L_0 (mm)	87.5
Young's modulus E (MNm^{-2})	206000
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Design	N	d	β_0	Volume
Opt.1	18	0.266	31.54	670



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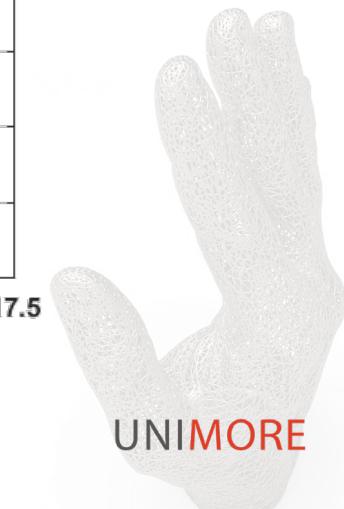
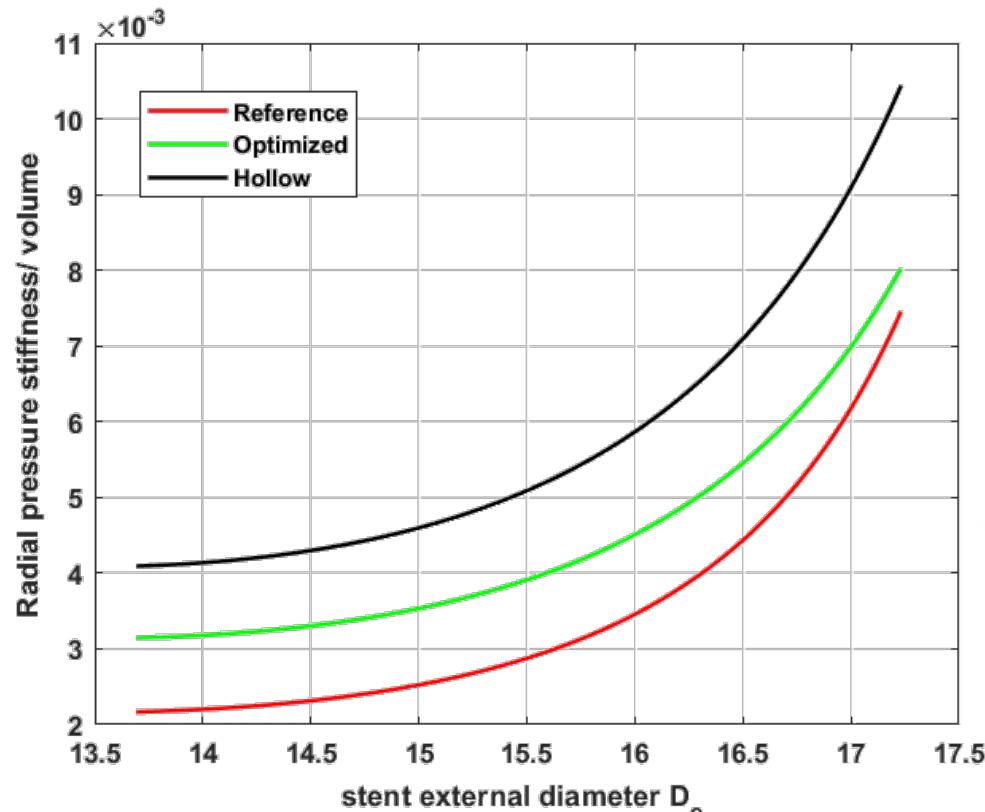
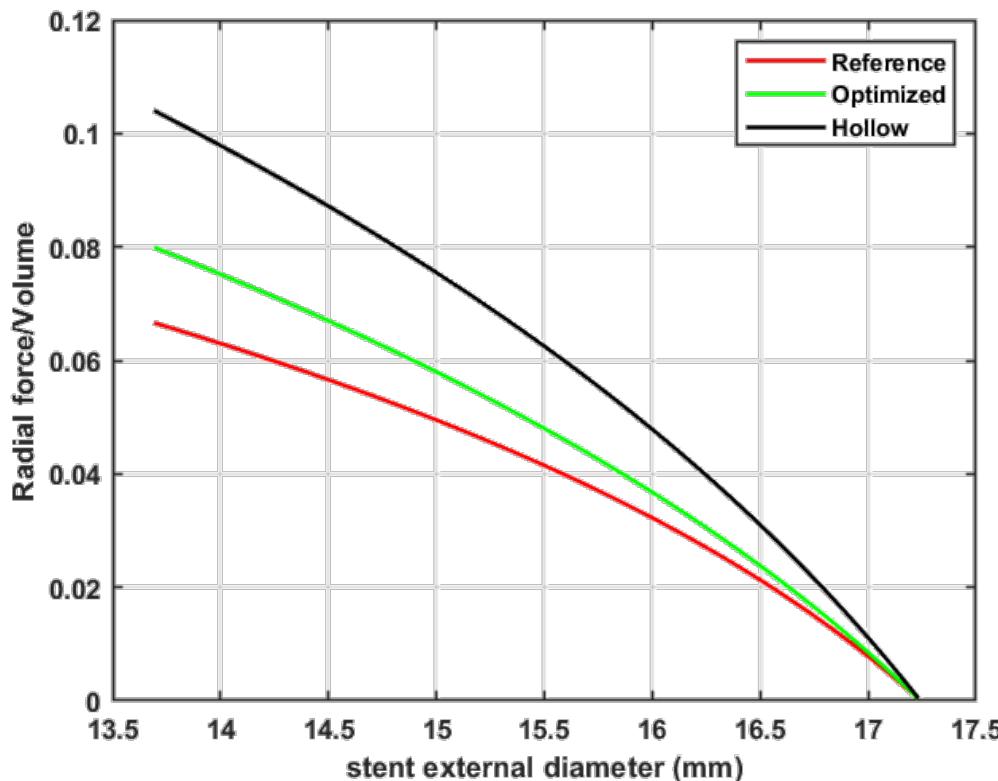
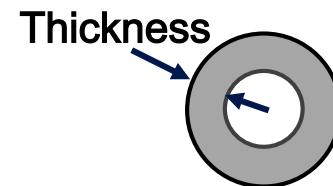


AM TECHNOLOGY : 3D PRINTED HOLLOW WIRES

$$\text{Volume Reduction Ratio (VRR)} = \frac{\text{initial volume} - \text{final volume}}{\text{initial volume}} \times 100$$

Thickness= 60 μm

Final volume= 468 mm^3 → **VRR = 30.17**



CONCLUSION & FUTURE DEVELOPMENT

1. Optimize the performance of an openended braided stent
2. Analytical approach + MOPSO optimization
3. Specific radial force F_r , specific radial pressure stiffness K_p , are optimized
4. F_r and K_p increased especially for lower diameters
5. The stent's maximum length is reduced from 51% to 1.41%.
6. A 3D-printed hollow structure is envisioned
7. According to the optimization stent's weight is lowered 30%.
8. Future steps: extend the optimization to superelastic NiTi

